SPECIFICATION AMENDMENTS

Please amend the Specification as follows:

Summary of the invention

Page 2, Paragraph 2:

The aforementioned object is achieved by ~~the features of claims 1 and 2.~~ utilizing a seal for a roller bearing having a sealing plate or cap attached to a rotating hub.

Page 2, Paragraph 3, bridging over to page 3:

In one embodiment, the present ~~The~~ invention ~~as claimed in claim 1~~ includes as seal a sealing plate which is fixed at least with a positive interlock to the rotating hub, which externally encloses the roller bearing. For this purpose a separate sealing medium, which effectively seals a sealing gap formed between the sealing plate and the hub, is provided between these components. When the sealing plate is fitted, the elastic seal lip or the separate sealing medium is tightly joined to the housing. For this purpose the sealing plate on the inner side in the area of the central opening has an elastic seal lip or a separate sealing medium, which serves to seal the annular gap between the rotationally fixed housing and the sealing plate. The elastic seal at least is advantageously designed so that a bracing force of the rotating elastic seal fixed in position on the hub diminishes as the rotational speed increases. This principle decisively reduces the heat generated by the seal and improves the service life of the roller bearing.

Page 3, Paragraph 2:

In another embodiment, the present ~~The~~ invention ~~as claimed in claim 2 as seal~~ comprises a sealing cap, which is secured both by positive interlock and by force-locking to the rotating hub enclosing the outer bearing ring. The sealing cap extends over the entire front face of the roller bearing. A separate elastic sealing medium is provided, in order to seal off a sealing gap formed between the sealing cap and the hub.

Page 4, Paragraph 1:

The advantageous ~~Advantageous~~ developments of the ~~inventions from the subject matter of the dependent claims 3 to 21.~~ present invention, together with other important aspects thereof, will be further appreciated upon reading the detailed description in conjunction with the drawings.

Page 6, Bridging Paragraph 3:

A preferred development of the invention ~~as claimed in claim 1~~ relates to the design of the elastic seal on the inside in the area of the central opening in the sealing plate. The seal lip of this seal is in this case provided at the end with an external bead, which forms an additional mass. For this purpose the bead is arranged so that a centrifugal force acting at the center of mass of the bead gives rise to a component force, which continuously lifts off from the supporting face as the rotational speed increases. This serves to reduce the bracing force or the contact force of the seal lip on the associated cylindrical section of the housing. This measure produces a good sealing property when the internal combustion engine is at rest, in that the seal lip bears on the corresponding contact face with maximum bracing force. When the seal is rotating, that is to say when the internal combustion engine is running, for example, the centrifugal force causes the bracing force of the

seal lip to diminish continuously as the rotational speed increases. The seal lip of beaded design serves to exert a direct influence on the working of the seal. In particular, the bearing force of the seal lip can be influenced by the mass of the bead and the axial distance of the bead from the radial flank or a center of rotation of the seal lip. The displacement of the seal lip under centrifugal force has the advantage that in the operating state, especially at high rotational speeds, the rubbing contact that occurs between the seal lip and the associated contact face is reduced to the extend that there is no detrimental friction associated with the generation of heat. With the internal combustion engine and the roller bearing at rest, on the other hand, a good sealing property occurs which will, for example, pass even an extreme test, complete immersion in a water bath.

Page 9, Paragraph 2:

According to the invention a lubricant ~~SF507 or KE2/3K-10 conforming to DIN 51 502~~ in accordance with the American Petroleum Institute Service Symbol SF 507 or the Designation of Lubricants Standard DIN 51502 KE 2 K-10 or DIN 51502 KE 3 K-10 as are commonly known in the art is preferably used as elastic medium. The code designation in accordance with DIN 51502 provides information about the type of lubricating grease, the suitability, consistency classes and the application temperature range of lubricants commercial grease products conforming to these trade standards are available under various trademarks. However, the invention is not limited to this but likewise encompasses other suitable alternative elastic media.